# TWENTIETHANNUAI

estConX

#### March 3 - 6, 2019

Hilton Phoenix / Mesa Hotel Mesa, Arizona

Archive

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Session 8 Presentation 3

## TestConX 2019

Running Hot & Cold - Thermal Management

# Closer Tolerance Thermal Management at the Device-Under-Test

#### Bert Brost and Mehdi Attaran Cohu



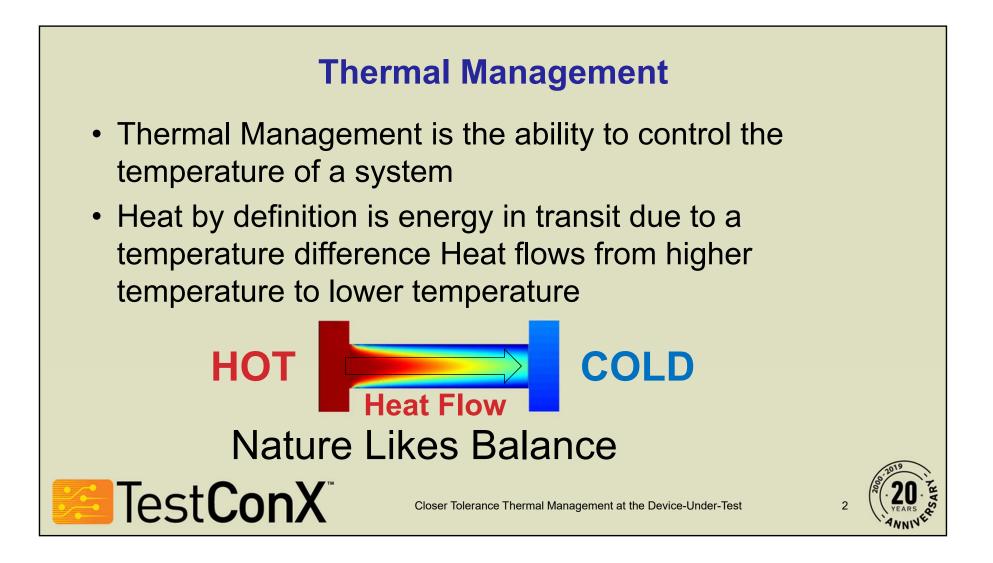


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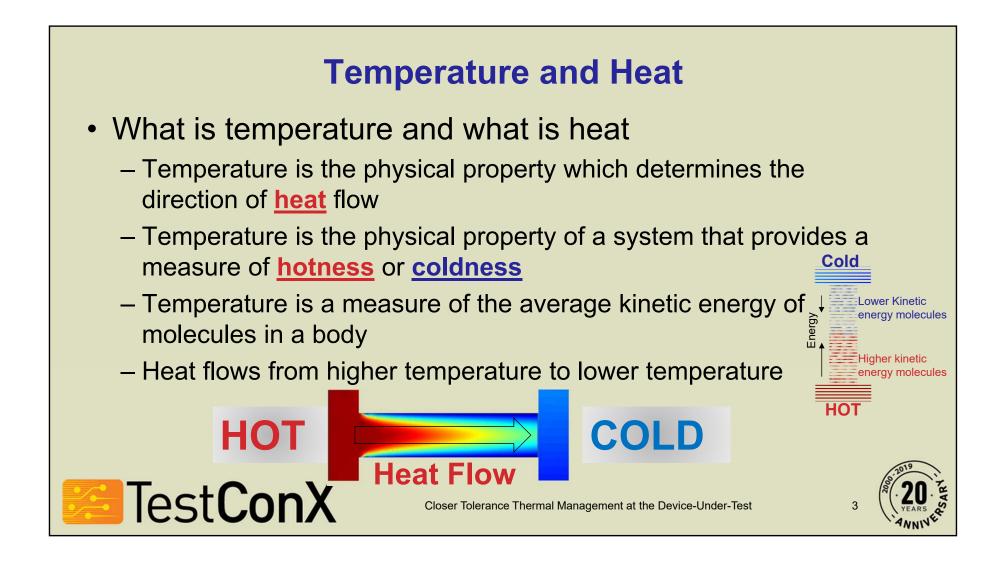
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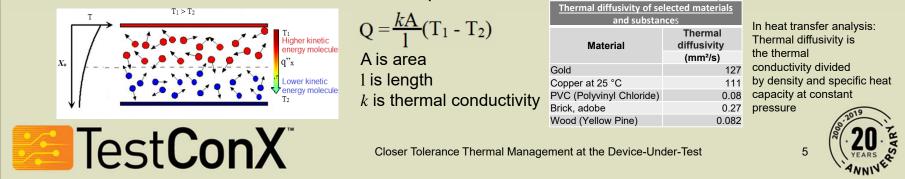


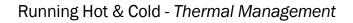
<ul> <li>Analogies</li> <li>Electrical engineers use Ohm's law as an analogy when doing thermal resistance calculations</li> <li>Mechanical and structural engineers use Hooke's Law and use it as an analogy when doing</li> </ul>			
<u>ce calculations</u> Structural Analog	Hydraulic Analogy	Thermal	Electrical
	Volume V [m <sup>3</sup> ]	Heat Q [J]	Charge q[C]
Displacement X[m]	Pressure P[n/m <sup>2</sup> ]	Temperature T[K=j/k <sub>b</sub> ]	Potential V[V=J/C]
Load of force F [N]	Flow rate Q [N/m <sup>3</sup> /s]	Heat Transfer rate Q [W=J/s]	Current I [A=C/s]
Stress σ[Pa=N/m <sup>2</sup> ]	Velocity <b>v</b> [m/s]	Heat Flux q [W/m <sup>2</sup> ]	Current Density <b>j</b> $[C/(m^2 \cdot s)=A/m^2]$
Flexibility 1/k []	Fluid resistance R[]	Thermal Resistance R [K/W]	Electrical Resistance R [Ω]
Stiffness k [N/m]		Thermal Conductivity 1/R [W/(K·m)]	Electrical Conductance 1/R []
<u>Hooke's Law</u> ΔX=F/k	Hagen-Poiseuille Equation P=QR	Newton Law of Cooling T=QR	Ohms Law V=IR
		<u>Fourier's law</u> O=-k∇T	Ohm's law J=σE
	structural engines ce calculations Structural Analog Displacement X[m] Load of force F [N] Stress σ[Pa=N/m <sup>2</sup> ] Flexibility 1/k [] Stiffness k [N/m] <u>Hooke's Law</u>	structural engineers use Hooke's Law         Structural Analog       Hydraulic Analogy         Structural Analog       Volume V [m³]         Displacement X[m]       Pressure P[n/m²]         Load of force F [N]       Flow rate Q [N/m³/s]         Stress σ[Pa=N/m²]       Velocity v[m/s]         Flexibility 1/k []       Fluid resistance R[]         Stiffness k [N/m]       Hagen-Poiseuille	structural engineers use Hooke's Law and use it as an arstructural AnalogHydraulic AnalogyThermalStructural AnalogHydraulic AnalogyThermalVolume V [m³]Heat Q [J]Displacement X[m]Pressure P[n/m²]Load of force F [N]Flow rate Q [N/m³/s]Stress $\sigma$ [Pa=N/m²]Velocity $\mathbf{v}$ [m/s]Heat Flux q [W/m²]Flexibility 1/k []Fluid resistance R[]Thermal Resistance R [K/W]Stiffness k [N/m]Thermal Conductivity 1/R [W/(K·m)]Hooke's Law $\Delta X=F/k$ Hagen-Poiseuille Equation P=QRFourier's lawFourier's law

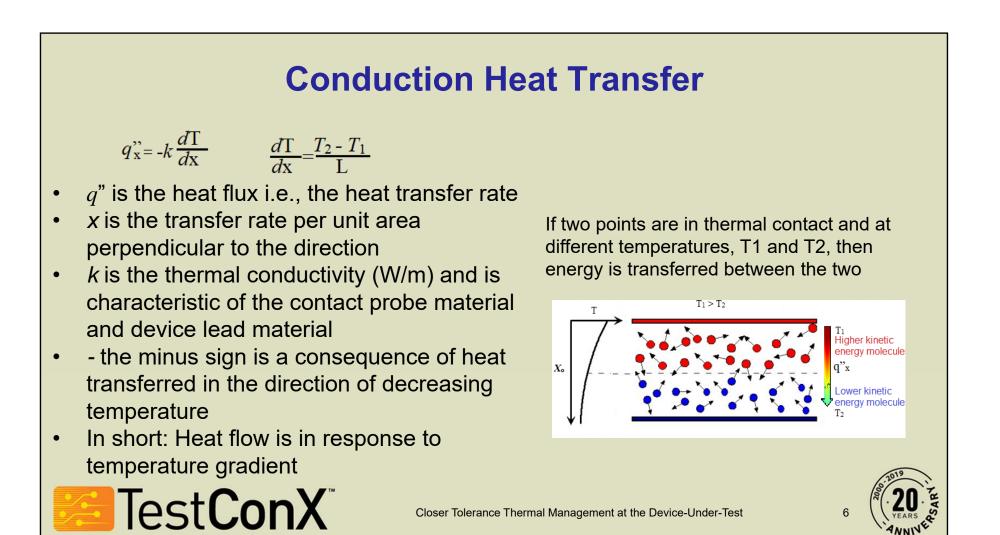
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### **Heat Transfer**

- · Heat flows in response to a temperature gradient
- If two points are in thermal contact and at different temperatures, (e.g., T 1 and T 2) then energy is transferred between the two in the form of heat, Q
- The rate of heat flow from T1 to T2 depends on the two temperatures and the material conductivity
- If heat flows from hot to cold, (the standard convention)







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### **Forced Convection Heat Transfer**

Convection is heat energy transferred between a surface and a moving fluid with different temperatures as convection.

A fluid is anything that has loosely moving molecules that can move easily from one place to another, i.e., Liquids and gases

Though the convection thermal transport is more complicated than with conduction, convection heat transfer is complicated since it involves fluid motion as well as heat conduction. The fluid motion enhances heat transfer (the higher the velocity the higher the heat transfer rate). Fluid motion and convection go beyond the scope of the paper.

The basic convection heat transfer equation is quite simple:  $q = k A \Delta T$  (Fourier's Law)

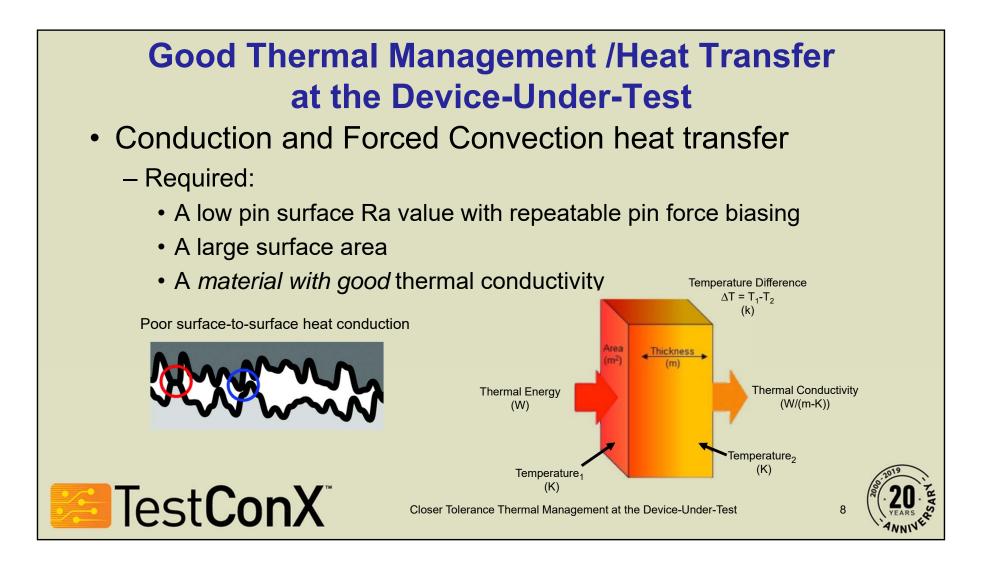
- *q*" is the heat flux i.e., the heat transfer rate (*Q* is heat transport per unit of time)
- *k* is the thermal conductivity (W/m) and is characteristic of the contact probe material and device lead material
- A is the heat transfer area of the surface (m<sup>2</sup>)
- $\Delta T$  is the difference between the surface and the bulk fluid (°C)



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#### cDragon

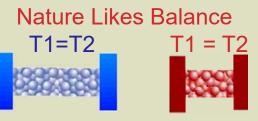
- Thermal
  - Multi beam design allows for better airflow thru contact pin elements
    - Integrated airflow tunnels allow the pins to remain very close to handler's set temperature during testing
    - Heat by definition is energy in transit due to a temperature difference

#### Conduction $Q = \frac{kA}{l}(T_1 - T_2)$

- Q is heat transport per unit of time
- A is area
- 1 is length
- *k* is thermal conductivity







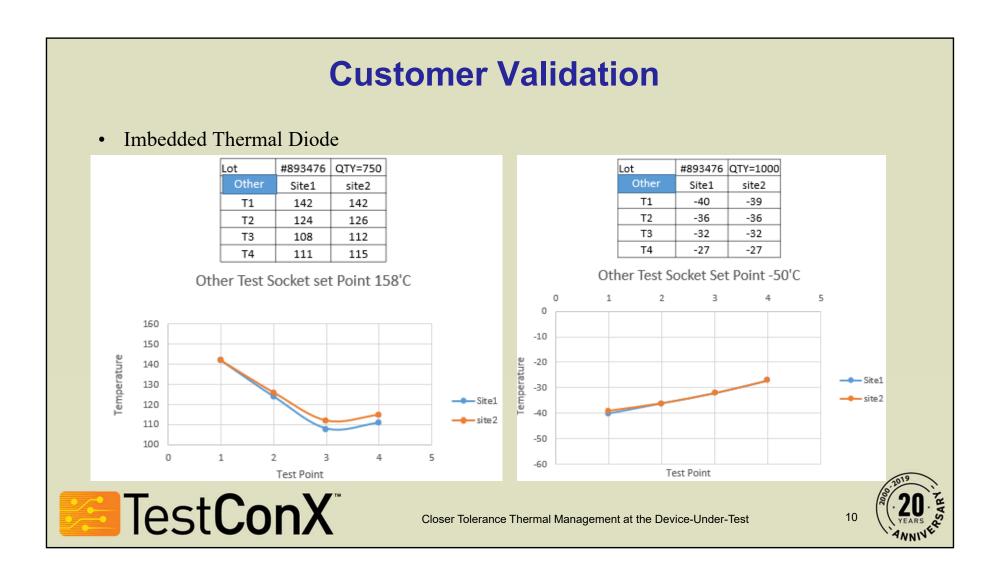
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## $\boldsymbol{q} = \overset{\text{Convection}}{\boldsymbol{k}} \boldsymbol{A} \, \Delta \boldsymbol{T}$

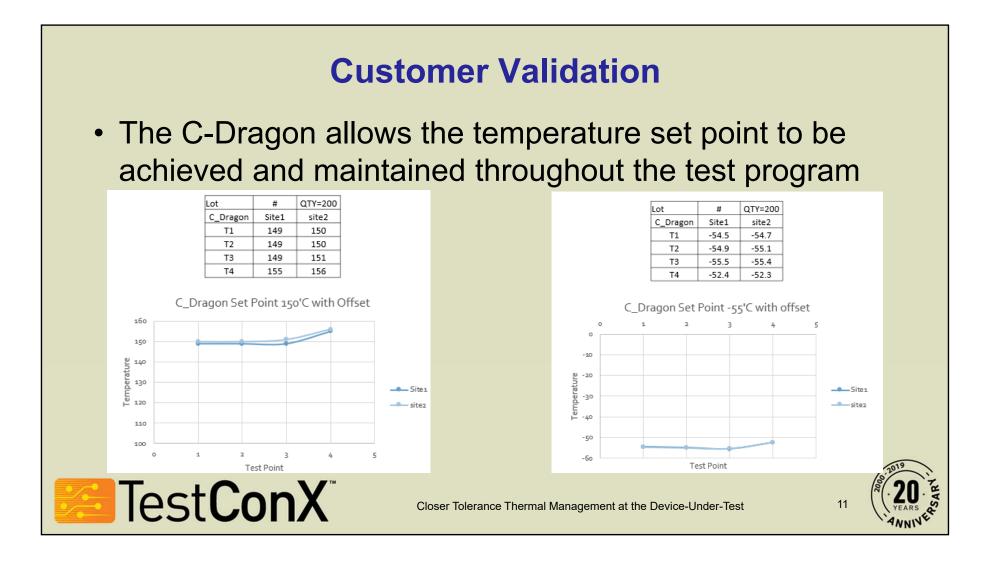
- *q*" is the heat transfer rate *k* is the thermal conductivity
- A is the heat transfer area of the surface (m<sup>2</sup>)
- $\Delta T$  is the difference between the surface and the bulk fluid (°C) temperature



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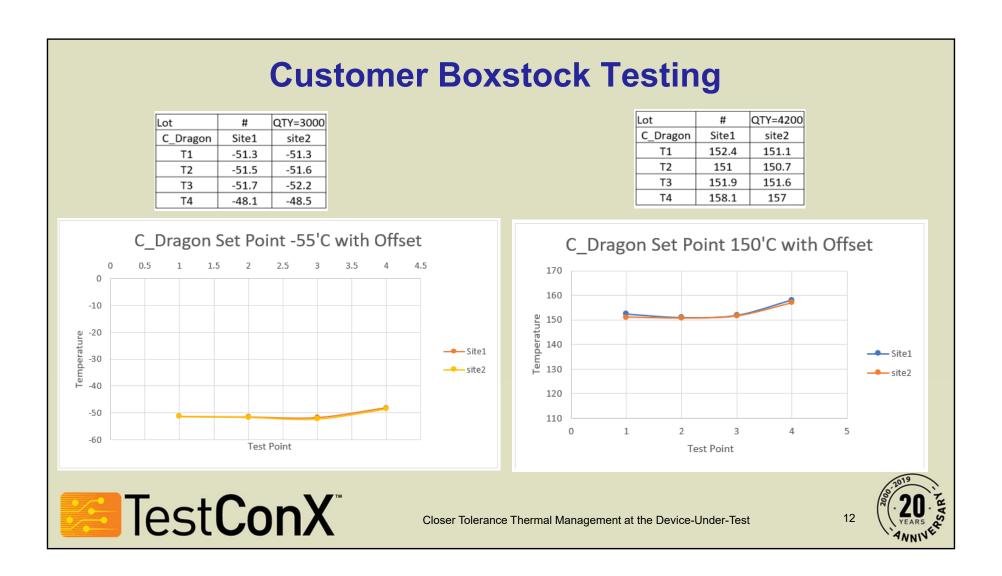


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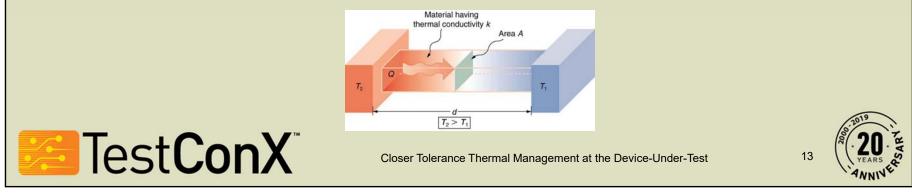
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### Conclusion

- With the use of Thermal Conduction physics, a contactor with better heat response time and Temperature set point accuracy can be developed and deployed
- This was proven in the laboratory and again on the test floor.
- By design, heat transfer in a contactor takes place in two modes: convection and conduction
- Thermal Resistance is analogous to Electrical Resistance in that it is inversely proportional to the flow of heat.



#### References

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